## CS5643

05 Mass-and-spring models

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## Binary Spring

## The most basic modeling tool for all kinds of deformable things

## Spring defined by

- which particles $i$ and $j$ it connects
- its spring stiffness $k_{s}$
- its rest length $l_{0}$


From Hooke's law we know force is proportional to displacement from rest

- $f=k_{s}\left(l-l_{0}\right)$

The force acts along the direction of the spring

- $\mathbf{f}_{i}=k_{s}\left(\left\|\mathbf{x}_{i j}\right\|-l_{0}\right) \hat{\mathbf{x}}_{i j}$ where $\mathbf{x}_{i j}=\mathbf{x}_{j}-\mathbf{x}_{i}$ (quick sanity check: pulls $i$ towards $j$ when stretched)
- $\mathbf{f}_{j}=k_{s}\left(\left\|\mathbf{x}_{j i}\right\|-l_{0}\right) \hat{\mathbf{x}}_{j i}=-\mathbf{f}_{i j}$


## Adding damping

## Springs are usually too springy!

- a system of springs will oscillate forever...


## Damping will dissipate energy

- but using the drag forces $\mathbf{f}_{d}=-k_{d} \mathbf{v}$ we use for free particles can slow things inappropriately
- this force opposes all motion; we only want to oppose the spring's movement

Spring damping force opposes changes in spring length only

- only opposes relative motion
- only opposes motion in the direction of the spring

- $\mathbf{f}_{i}=k_{d}\left(\mathbf{v}_{i j} \cdot \hat{\mathbf{x}}_{i j}\right) \hat{\mathbf{x}}_{i j}$ where $\mathbf{v}_{i j}=\mathbf{v}_{j}-\mathbf{v}_{i}$ (sanity check: pulls $i$ towards $j$ when elongating)


## Phenomena of damped springs

## Oscillations of undamped springs

- a 1D damped spring obeys $m \ddot{x}+k_{d} \dot{x}+k_{s} x=0$
- when $k_{d}$ is negligible the solution is $x(t)=C_{1} \cos \left(\omega t+C_{2}\right)$ where $\omega^{2}=\frac{k_{s}}{m}$
- stiffer spring $\rightarrow$ faster oscillation; higher mass $\rightarrow$ slower oscillation
. when $k_{s}$ is negligible the solution is $x(t)=c_{0}+c_{1} T\left(1-e^{t / T}\right)$ where $T=\frac{m}{k_{d}}$
- higher mass $\rightarrow$ takes longer to stop; higher damping $\rightarrow$ takes less time to stop
- in general case we get $x(t)=C e^{-t / T} \cos \left(\omega t+C_{2}\right)$ where

$$
T=\frac{2 m}{k_{d}} \text { and } \omega=\sqrt{k_{s} / m-(1 / T)^{2}}
$$

- decaying oscillation combining the two above behaviors
- damping also slows the oscillation; when we reach $\omega=0$ the system is "critically damped"



## Modeling rods with springs

## A rod is a long, slender, flexible object (essentially 1D)

- can stretch or bend and elastically resist both


## Basic plan: a chain of masses and springs

- $N$ springs of length $L / N$ and spring constant $\ldots N k_{s}$ (why?)

- this handles stretching but doesn't oppose bending-too "floppy" and chain-like


## Simple way to resist bending: bending springs

- springs that skip one particle
- $N-1$ springs of length $2 L / N$ and spring constant $N k_{s}^{\prime}$ where $k_{s}^{\prime}$ is usually quite a bit less than $k_{s}$



## Modeling cloth with springs

## Very similar idea to rods, but in a 2D grid

- structural springs along the axes (stiff for woven cloth, softer for knitted)
- bending springs skipping one particle (weak to allow lots of bending)
- shear springs along the diagonals (weaker than structural, to allow shearing)


## You'll do this in the assignment!

- it's only cloth if it's in 3D, so can't really demo this ;)



## Indexing in a spring mesh

## index particles in 2D

- avoids row-column indexing calculations


## index springs the same way

- think of all springs connected to one particle
- compute forces by visiting all particles and considering all springs connected to that particle
- (to enumerate the springs, visit particles and considering only the springs connected to later particles)


## Modeling deformable solids with springs

## 2D: looks a lot like cloth

- don't need bending springs
- shear springs should probably be stronger than for cloth
- a triangular mesh only requires springs along the edges
- in a 2D space, a 2D object can resist compression



## 3D: need enough springs to prevent collapsing

- for a cube mesh, various strategies are possible - bracing diagonals of faces, or bracing across the diagonals of the cube
- a tetrahedral mesh is naturally stable with just a spring along each edge


## Deriving forces from energies

## Binary springs are simple and are a lot of fun to play with but they eventually start to become limited

- bending and shear springs contribute also to stretching stiffness
- difficult to achieve behavior matching particular measurements or material models
- bending springs are not very good at resisting slight bending
(bending stiffness $=0$ when straight!)
- difficult or impossible to express things like volume or area preservation


## The spring force belongs to a useful class

- it is a conservative force, meaning it takes the same amount of work to get from one configuration to another regardless of the path
- this means it is the derivative (gradient in this case) of a potential ... and the potential is literally the potential energy stored in the spring!


## Working out our spring force from the energy

## Start with the spring energy

- $E_{i j}(\mathbf{x})=\frac{1}{2} k_{s}\left(\left\|\mathbf{x}_{i}-\mathbf{x}_{j}\right\|-l_{0}\right)^{2}$ (this is the contribution of one spring to the total system energy)


## Force is minus the gradient of energy

$\mathbf{f}_{i}(\mathbf{x})=\frac{\partial E}{\partial \mathbf{x}_{i}}(\mathbf{x})$ (remember $\mathbf{x}$ is a big vector of all the positions; this partial derivative is zero for all the particles that are not connected to this particular spring)

## Take the computation one step at a time:

- derivative of $\mathbf{x}_{i}-\mathbf{x}_{j}$
- derivative of $\|\mathbf{v}\|$ wrt. $\mathbf{v}$
- derivative of $E_{i j}$ wrt $\|\mathbf{v}\|$

